



University of  
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# Mechanics of Solids

## MMME2053

### Shear Stresses

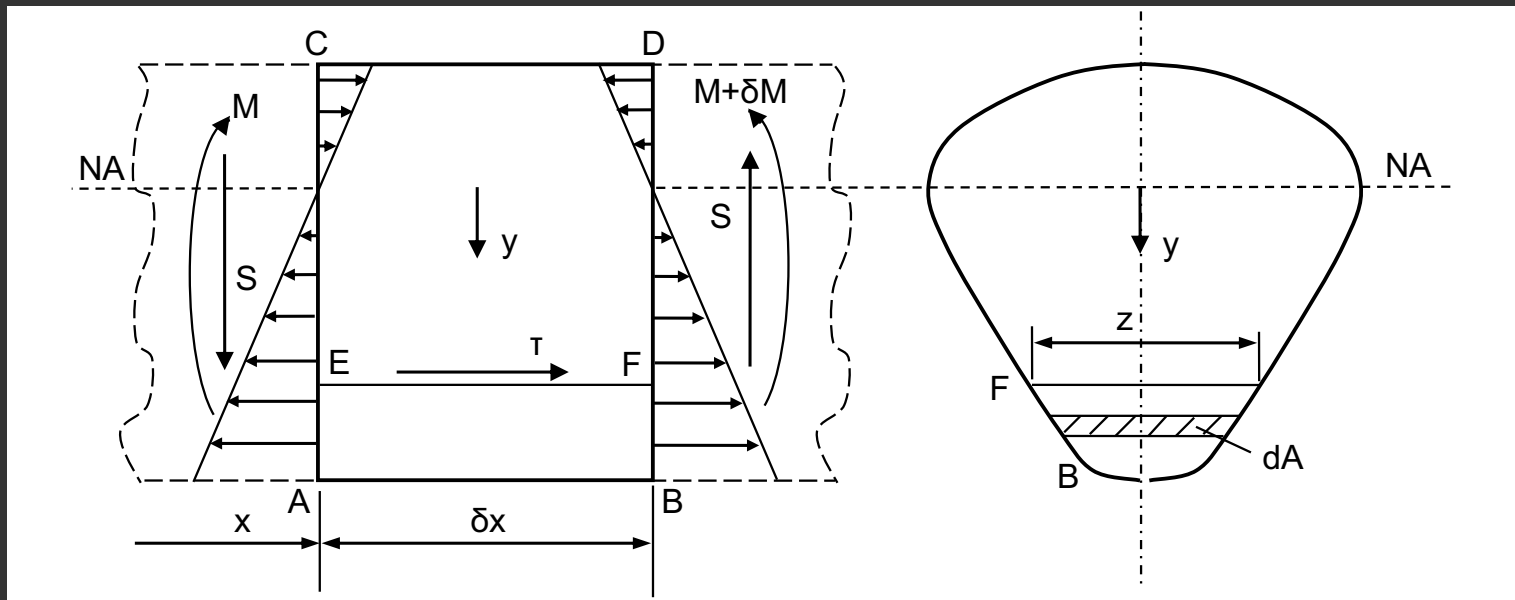
### Lecture 2

# Shear Stresses in Beams (1)

- The through-thickness shear force in a beam is the integral of the shear stresses over the cross-section
- We will derive an expression for the transverse (through thickness) shear stress at any position of a section in an arbitrary beam as a function of the shear force

# Shear Stresses in Beams (2)

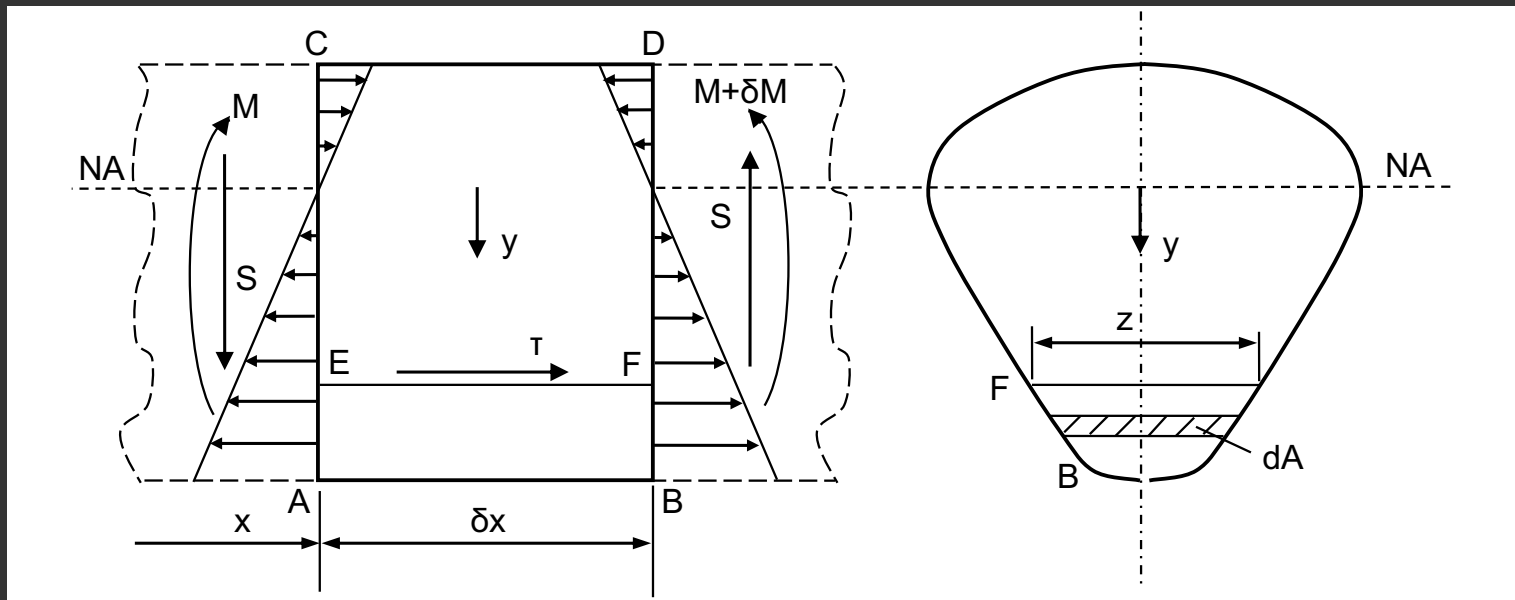
- Consider an element of beam length,  $\delta x$ , as shown below. The bending moment at  $x$ , section AC, is  $M$  and at  $x + \delta x$ , section BD, is  $M + \delta M$ .



- The direct bending stresses on AC are: 
$$\sigma_{AC} = \frac{My}{I}$$

# Shear Stresses in Beams (3)

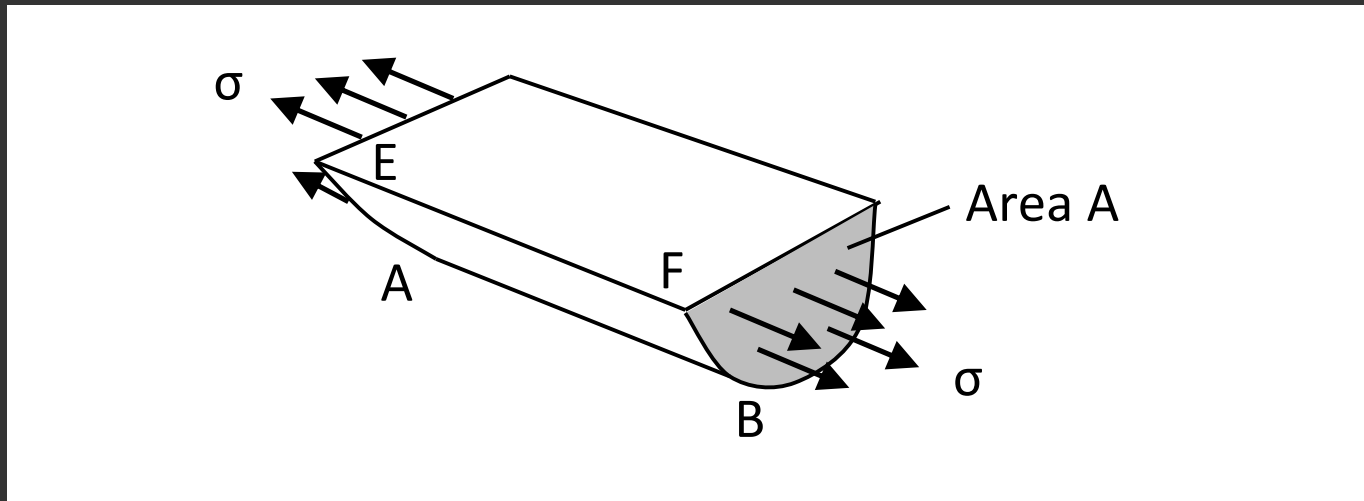
- And the direct bending stresses on BD are given by



$$\sigma_{BD} = \frac{(M + \delta M)y}{I}$$

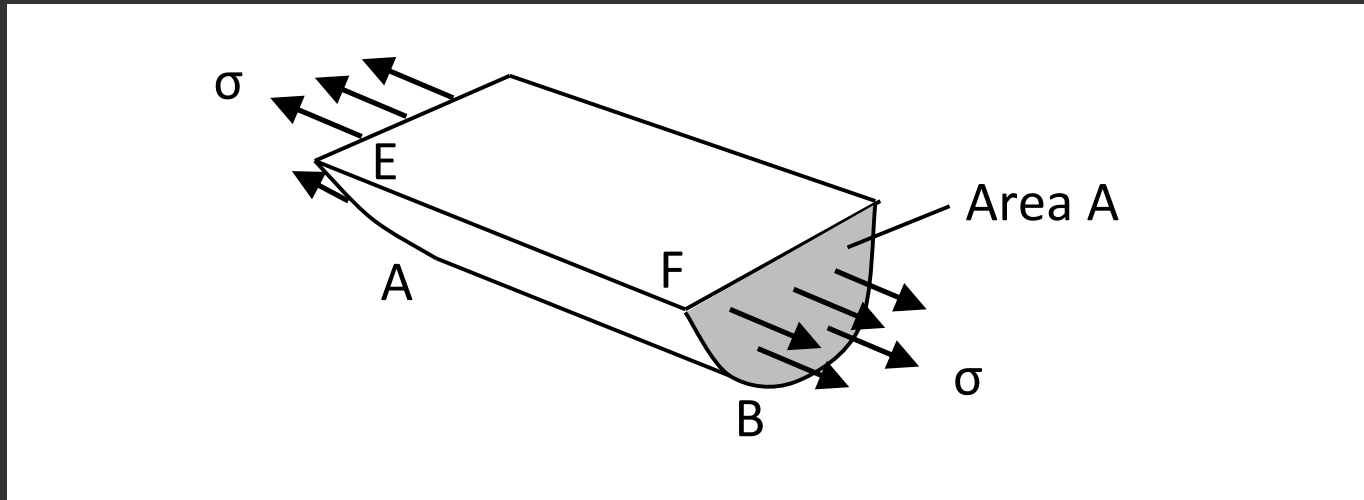
# Shear Stresses in Beams (4)

- Thus, when the bending moment varies along the length of the beam on an element such as ABEF, there is a net axial force due to change in the bending stresses



# Shear Stresses in Beams (5)

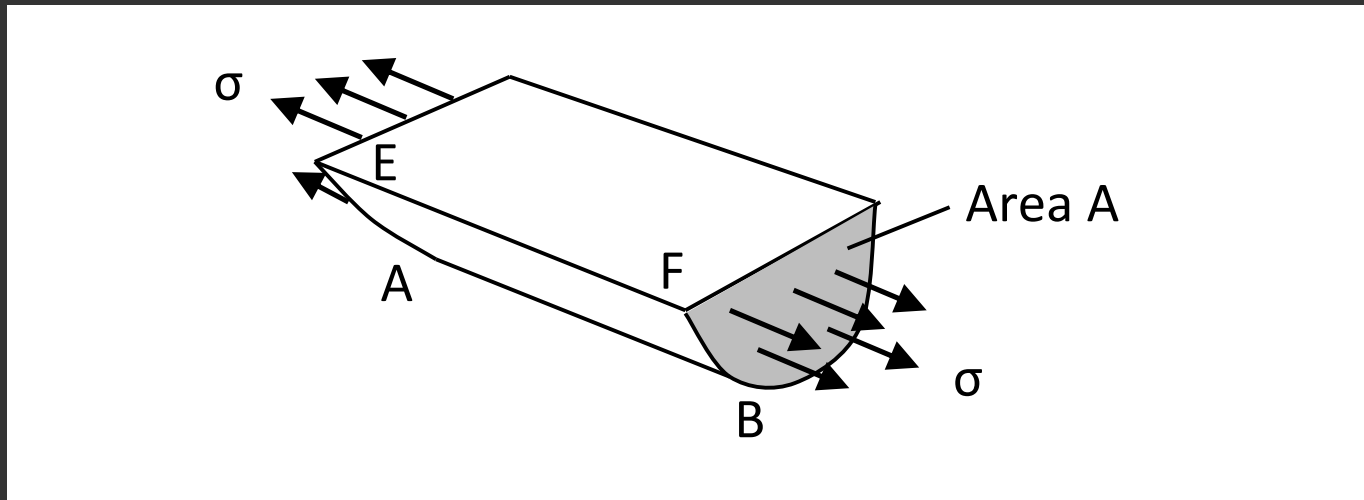
- The force on the face EA is the integral of the bending stresses over the area,  $A$



$$F_{EA} = \int_A \frac{M}{I} y dA$$

# Shear Stresses in Beams (6)

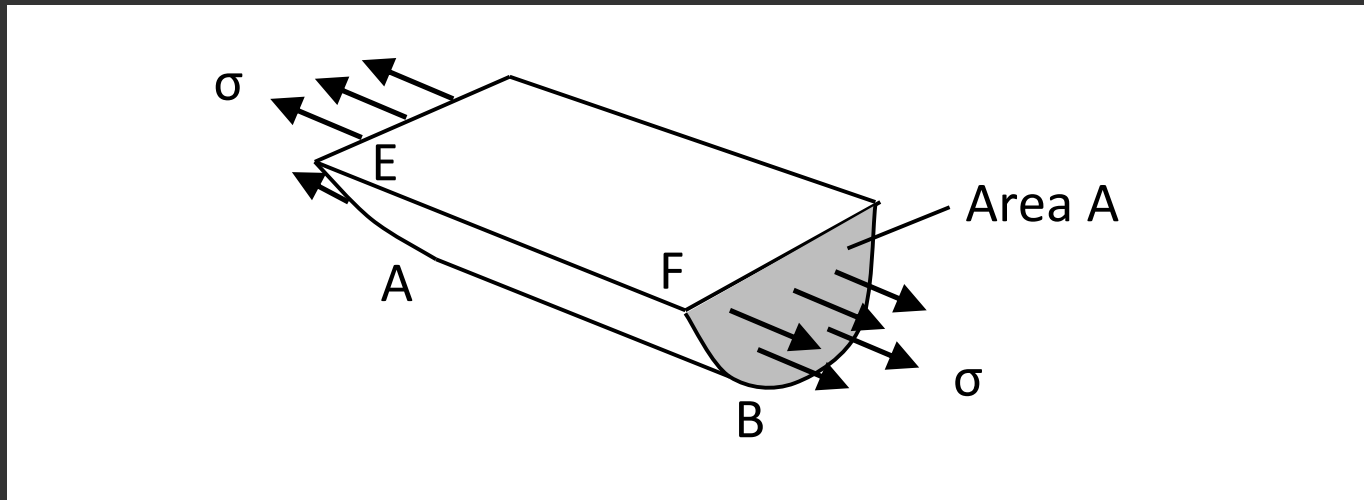
- The force on the face FB is the integral of the bending stresses over the area,  $A$



$$F_{FB} = \int_A \frac{(M + \delta M)}{I} y dA$$

# Shear Stresses in Beams (7)

- The net force to the right acting on the element ABEF is the difference in these

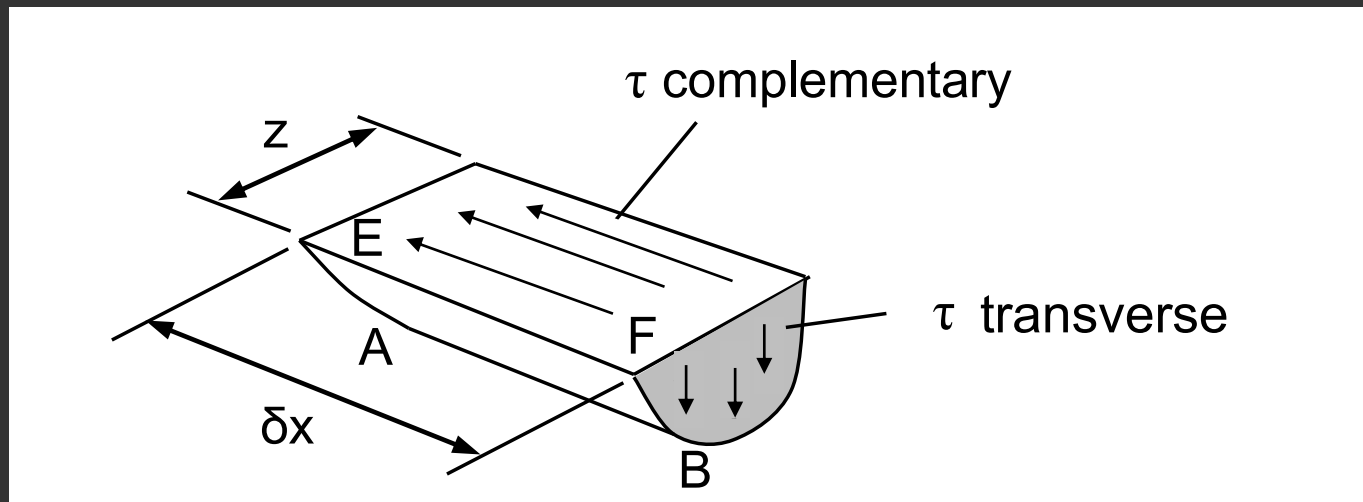


$$\text{Net Force} = \int_A \frac{\delta M}{I} y dA \quad [1]$$



# Shear Stresses in Beams (8)

- In order to maintain equilibrium of ABEF, shear stresses must act on the plane EF, of average value  $\tau$ . These shear stresses are complementary to the transverse shear stresses



- The net force to the left due to these complementary shear stresses is:

$$\text{Net Force} = \tau z \delta x$$

# Shear Stresses in Beams (9)

- Equilibrium of ABEF requires the net force due to bending to balance the net force due to the complementary shear

$$\tau z \delta x + \int_A \frac{\delta M}{I} y dA = 0$$

$$\tau z \delta x = \frac{1}{I z} \frac{\delta M}{\delta x} \int_A y dA$$

# Shear Stresses in Beams (10)

- In the limit,

$$\lim_{\delta x \rightarrow 0} \frac{\delta M}{\delta x} = \frac{dM}{dx} = -S$$

- Where  $S$  is the shear force at the section

# Shear Stresses in Beams (11)

- Which gives:

$$\tau = \frac{S}{Iz} \int_A y dA \quad [2]$$

- This is the general integral expression for transverse shear stress at any position  $y$  through the thickness.

# Shear Stresses in Beams (12)

- Which can be written in discrete form as:

$$\tau = \frac{SA\bar{y}}{Iz} \quad [3]$$

- where  $A$  is the area of the part of the cross-section outside the position at which  $\tau$  is determined, and  $\bar{y}$  is the distance of the centroid of this area from the neutral axis.

# Shear Stresses in Beams (13)

- May see it in some texts as:

$$\tau = \frac{SQ}{Iz}$$

- Where  $Q$  represents  $A\bar{y}$  but is generally more applicable for complex sections with changes in cross-sectional area through the depth of the beam. We can calculate  $Q$  for each sub-area of the section and sum them together.

# Shear Stresses in Beams (14)

- For a general beam cross section:

$$\tau = \frac{SA\bar{y}}{Iz}$$

