

# Mechanics of Solids

### **Thermal Stress and Strain** Worked Example 2

A rectangular beam, of width b and depth d, has a temperature variation given by:



There is no restraint or applied loading, so P = M = 0. Obtain the stress distribution

#### Axial Force Equilibrium

Equation (5) 
$$P = E\bar{\varepsilon}A - E\alpha \int_A \Delta T dA$$

so, 
$$0 = E\bar{\varepsilon}bd - E\alpha \int_{-\frac{d}{2}}^{\frac{d}{2}} \Delta T_{max} \left(1 - \frac{4y^2}{d^2}\right)bdy$$

Axial Force Equilibrium  
Rearranging, 
$$\bar{\varepsilon} = \frac{\alpha}{d} \Delta T_{max} \int_{-\frac{d}{2}}^{\frac{d}{2}} \left(1 - \frac{4y^2}{d^2}\right) dy$$

$$\bar{\varepsilon} = \frac{\alpha}{d} \Delta T_{max} \left[ y - \frac{4y^3}{3d^2} \right]_{-\frac{d}{2}}^{\frac{d}{2}}$$

$$\bar{\varepsilon} = \frac{2}{3} \alpha \Delta T_{max}$$

#### Moment Equilibrium

Equation (7) could be used to determine 1/R using M = 0

$$M = \frac{EI}{R} - E\alpha \int_{A} \Delta T y dA$$

But from symmetry, we can see that 1/R = 0 (radius is infinite)

#### **Stress Distribution**

Equation (3) 
$$\sigma_{\chi} = E\left(\bar{\varepsilon} + \frac{y}{R} - \alpha\Delta T\right)$$

Substituting in for  $\, \overline{\!\mathcal{E}} \,$  , 1/R and the given temperature variation gives

$$\sigma_{x} = E\left(\frac{2}{3}\alpha\Delta T_{max} + 0 - \alpha\Delta T_{max}\left(1 - \frac{4y^{2}}{d^{2}}\right)\right)$$

$$\sigma_x = E\alpha\Delta T_{max} \left(\frac{4y^2}{d^2} - \frac{1}{3}\right)$$

#### Evaluate Stress Distribution

At 
$$y = 0$$
  $\sigma_x = \frac{-E\alpha\Delta T_{max}}{3}$ 

At 
$$y = \pm d/2$$
  $\sigma_{\chi} = \frac{2E\alpha\Delta T_{max}}{3}$ 

#### Evaluate Stress Distribution

$$\sigma_x = 0 \text{ when } \frac{4y^2}{d^2} = \frac{1}{3}$$
 (from  $\sigma_x = E\alpha\Delta T_{max}\left(\frac{4y^2}{d^2} - \frac{1}{3}\right)$ )  
i.e.  $y = \sqrt{\frac{1}{12}d^2}$ 

or  $y = \pm 0.287d$ 

#### Evaluate Stress Distribution

Taking the points we know and plotting a transition between them gives:

